Pressure Control Valves

Back Pressure Regulators UV 1.5

Valve in Intermediate Flange Design



Technical Data

Description

Self-acting back pressure regulators are simple control valves offering accurate control while being easy to install and maintain. They control the pressure upstream of the valve without requiring pneumatic or electrical control elements.

The back pressure regulator UV 1.5 is a pilot-operated control valve with proportional control mode consisting of main valve and pilot valve. It is completely made of stainless steel with excellent corrosion resistance. Its intermediate flange design with limited size makes the valve extremely lightweight and compact. The valve cone has a metallic seal

The sturdy valve design and the metallic valve seal do not require any particular filtration of the operating fluid. Thanks to its medium-wetted movable components, the valve is largely maintenance-free. In addition, it can be installed in any desired mounting position.

Special notes:

The seal on the outlet side must not cover the outflow bore of the pilot valve (observe measurement!)

These valves are no shut-off elements ensuring a tight closing of the valve. In accordance with DIN EN 60534-4 and/or ANSI FCI 70-2 they may feature a leakage rate in closed position in compliance with the leakage classes II.

Standard

- » All stainless steel construction
- » FKM elastomers (O-rings)
- » 2014/68/EU Art. 4 Par. 3

Typical Applications

Maintaining the required lub oil pressure for the use of main and auxiliary oil pumps, for ex. compressors, gears, slide bearings, drive shafts etc., Pressure control of fuels / fuel oils in power plants. Control of minimum quantities for centrifugal pumps with oil / oily fluids. Lub oil systems, for ex. for steam and gas turbines, large diesel engines for ship propulsion and cogeneration units (CHP).

Operating instructions, know how and safety instructions must be observed. The pressure has always been indicated as overpressure. We reserve the right to alter technical specifications without notice.



K _{vs} -Values [m³/h]				
nominal diameter	DN 25	DN 50		
m³/h	10,5	19		

Setting Ranges [bar], Nominal Pressure				
Setting Range	1 - 20	12 - 40		
Nominal Pressure	PN 40	PN 40		

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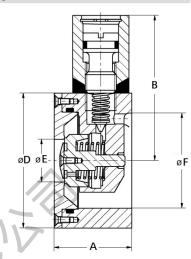
stainless steel 1.4404
stainless steel 1.4404 / 1.4462 / 1.4301
FKM
stainless steel 1.4310
stainless steel A4-70

Dimensions [mm]				
nominal diameter				
DN 25	DN 50			
40	50			
75	85			
70	100			
22	35			
49	69			
	DN 25 40 75 70 22			

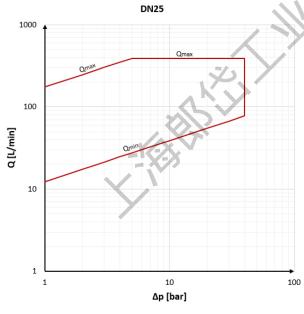
Flanges	
nominal diameter DN 25	nominal diameter DN 50
EN 1092-1 PN 40 DN 25, DN 32, DN 40 ASME B16.5 Class 300 NPS 1', NPS 1 - 1/2'	EN 1092-1 PN 40 DN 50, DN 65 ASME B16.5 Class 300 NPS 2', NPS 2 - 1/2'

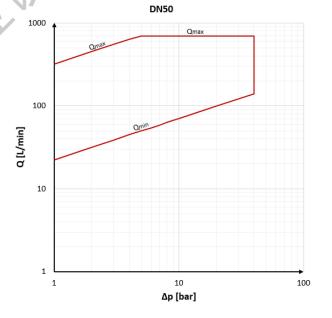
Weights [kg]		
nominal diameter	DN 25	DN 50
kg	1.0	2.4

Dimensional Drawing



Flow Chart





 $Q [m^3/h] = Q [L/min] / 1000 x 60$

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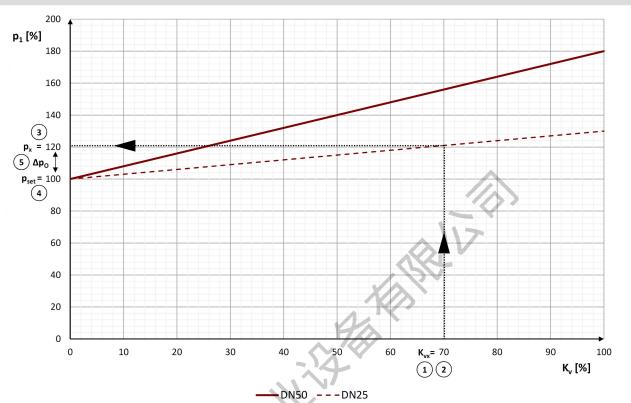
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Control Curve



p₁: Inlet Pressure
 p₂: Outlet Pressure
 Q: Operating volumetric flow

 K_{vx} : Flow coefficient at operating point K_{vs} : max. flow coefficient of the valve p_x : Upstream pressure at operating point

 p_{set} : set pressure of the valve Δp_0 : Opening pressure difference

Example

Determination of the opening pressure difference Delta Δp_0 for UV1.5 DN25

hydraulic oil ISO VG 46, T = 60 °C, Q = 23.6 m³/h, p_1 = 10 barg, p_2 = 0 barg, K_{vs} = 10.5 m³/h = 100% m³

1. $K_v = 7.3 \text{ m}^3/\text{h}$

2. $K_{vx} = K_v / K_{vs} \times 100 \% = 70 \%$

3. $p_x = p_1 = 10 \text{ barg} = 120 \%$

4. $p_{set} = p_1 / p_x x 100 \% = 10 barg / 120 \% x 100 \% = 8.3 barg$

5. $\Delta p_0 = p_x - p_{set} = 10 \text{ barg} - 8.3 \text{ barg} = 1,7 \text{ bar} = 20 \%$

The expected pressure drop from the operating point until the valve closes is 1.7 bar.

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Standing 06 03 2019 MANKENBERG

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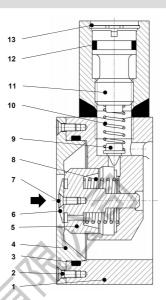
Function

To control the upstream pressure, the required set pressure at the valve is set at the adjusting screw (11). Use a common slot screwdriver for this purpose. Turning to the right increases the set pressure, turning to the left reduces the set pressure.

The piston chamber is fed via the gap at the baffle plate (6) and the control bore in the piston (5). In closed condition, the closing forces at the piston (5) prevail and keep the valve closed.

Once the pressure in the piston chamber goes beyond the set pressure, the cone (9) is lifted off the seat of the pilot valve against the pressure spring (10). The outflow to the outlet side causes a pressure drop in the piston chamber which lifts the piston (5) off the seat (4) and opens the back pressure regulator. The resulting balance of opening and closing forces keeps the piston (5) in position.

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Main Valve

- body
- countersunk head screw
- O-ring
- 4 seat
- piston
- baffle plate
- countersunk head screw
- 8 pressure spring

Pilot Valve

- 9 cone
- 10 pressure spring
- 11 set screw
- 12 O-ring

13 circlip

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